

Duration: 3 Hours

Total Marks: 80

- N. B. (i) Question number **one is compulsory**.
(ii) Answer any **three** questions from the rest.
(ii) Assume suitable data wherever necessary.

Q. 1 Write short note on any **four**

20

- Design Pressure and Design Temperature
- Various types of heads
- Power Requirement for Agitation
- Calendria Evaporator
- Stresses in Column Shell

Q.2(a) Design storage tank for following data: (Shell plates and bottom plates)

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Tank diameter = 3 m

Tank ht = 6 m

Density of liquid = 980 kg/m³

Superimposed load = 1200 N/m²

MoC = CS

Permissible stress = 95 N/mm²

Density of MoC = 7800 kg/m³

Corrosion allowance = 2 mm

E = 2*10⁵ N/mm²

Weld joint efficiency = 0.85

Shell plate and bottom plate size = 5000 x 2000 mm (L x W)

Q.2(b) Write design procedure for determination of shell thickness at different heights for distillation column.

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Q3 (a) Design a propeller operating at 350 rpm speed in a vessel of 1200 mm diameter with following data

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Internal pressure in vessel = 0.3 N/mm²

Specific gravity of liquid in vessel = 1.1

Diameter of agitator = 300 rpm

Power Number = 0.9

Overhang of shaft from bearing support = 1500 mm

Shaft material = steel

Permissible shear stress = 50 N/mm²

Elastic limit in tensions = 250 N/mm²

Modulus of elasticity = 2x10⁵ N/mm²

Q 3(b) Explain types of supports with neat diagram. 10

Q 4(a) Design a U-tube heat exchanger for the following data- 12

Shell Side:

Design Pressure = 0.8 N/mm²

Permissible stress of shell material = 100 N/mm²

Standard torispherical head with knuckle radius = 6% crown radius

25 % cut segmental baffles

Gasket on shell side = flat metal jacketed asbestos filled

Gasket factor = 3.75

Gasket seating stress = 53 N/mm²

Tube side :

No of Tubes = 40

Tube outside diameter = 20 mm

Design pressure of tube side fluid = 2 N/mm²

Permissible stress for tube material = 120 N/mm²

Tube pitch = square 35 mm

Channel and channel cover MOC same as shell, Joint with tubesheet ring facing with 18 mm width

Gasket factor = 5.5

Gasket Seating stress = 126 N/mm²

Design i) Shell diameter and thickness ii) Flanged joint between shell and tube sheet iii)

Tube sheet thickness

Q 4(b) Explain Internal Parts of Packed column 08

Q 5(a) Write the design procedure for a Standard Vertical Short Tube Evaporator for the following data- 12

Design should include-

- (a) Diameter of tube sheet,
- (b) Calandria sheet thickness,
- (c) Tube sheet thickness,
- (d) Drum diameter and thickness

Q 5(b) Explain Various types of jackets with neat diagram and write design of plain jacket 08

Q 6(a) Design a pressure vessel subjected to internal pressure for the following data:

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Shell and head data: Operating pressure = 0.5 N/mm^2 ,

ID of the shell = 900 mm

Permissible stress for head and shell material = 95 N/mm^2 ,

Corrosion allowance = 1.5 mm

Weld joint efficiency = 0.85

Crown radius = Shell ID

Knuckle radius = 10 % of shell ID

Flanged joint: Gasket factor = 3.75,

Min gasket seating stress = 52.5 N/mm^2

Flange material same as shell material.

Permissible stress for bolt material = 138 N/mm^2

Design should include Shell, tori spherical Head, flanged joint.

Q.6(b) Short Note on

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i) Explain types of losses in storage of volatile liquids.

ii) What are standard and codes? List out different standards.