

(3 Hours)

Marks : 80

NB:

- (1) Question No. 1 is compulsory.
- (2) Solve any three questions out of remaining five questions
- (3) Draw figures wherever needed
- (4) Assume suitable data wherever necessary but justify the same.

Q1 Explain any four**20**

- a) Baffles and Tie rods
- b) Multiple Effect Evaporator
- c) Tray support
- d) Multishell and Shrink fit construction for High pressure vessel
- e) Growth crystalizer

Q 2 (a) Design a Fixed tubesheet heat exchanger for the following data**10**Design pressure = 0.6 N/mm²Permissible stress for shell material, Carbon steel = 100 N/mm²

Standard torispherical head with knuckle radius as 10 % of crown radius

25 % cut segmental baffles are provided

Number of tubes = 60

Tube outside diameter = 20

Design Pressure for tube side fluid = 21 N/mm²Permissible stress of tube material = 120 N/mm²

Tube pitch = Square

Flange material of construction – carbon steel

Permissible stress for flange material

Permissible stress for bolt material = 140 N/mm²

Mean Gasket Diameter = 428 mm

Gasket width = 24 mm

Gasket Factor = 3.75

Gasket seating stress = 53.4 N/mm²

Design

1. Shell Diameter and Shell Thickness
2. Flange thickness

- (b) Explain in detail any two types of shell and tube type of heat exchanger in view of differential thermal expansion between the shell and tubes. **10**
- Q3** (a) A thick cylinder with internal diameter of 40 mm and outside diameter as 80 mm is subjected to an internal pressure of 80 N/mm². Plot the stress distribution **10**
- (b) Discuss design procedure of making high pressure vessel based on various theories of failure **10**
- Q4** (a) Determine the wall thickness of the shell of a tall column with the following data **20**
- | | |
|--|-------------------------|
| Shell outside diameter | = 1500 mm |
| Vessel Height | = 15 m |
| Skirt height | = 3m |
| Design pressure | = 0.5 N/mm ² |
| Corrosion allowance | = 2 mm |
| Tray spacing | = 0.6 m |
| Top disengaging spacing | = 1m |
| Bottom separator Space | = 2 m |
| Specific gravity of carbon steel | = 7.7 |
| Insulation Thickness | = 50 mm |
| Density of Insulation | = 780 Kg/m ³ |
| Head 2:1 elliptical | |
| Weight of each head | = 5 kN |
| Wind velocity | = 125km/hr |
| Weight of liquid and trays | = 700 N/m ² |
| Permissible stress for shell wall material | = 100 N/mm ² |
- Q5** (a) Design a standard vertical short tube evaporator of calendria type for the following data **20**
- | | |
|--------------------------------------|--------------------------|
| External pressure on evaporator drum | = 0.10 N/mm ² |
| Amount of water to be evaporated | = 25000 N/hr |
| Heating surface required | = 225 m ² |
| Steam pressure | = 0.15 N/mm ² |
| Density of liquid | = 9850 N/m ³ |
| Density of vapor | = 0.84 N/m ³ |
| Materials used for the evaporator | = low carbon steel |

structure

Central down take = 40 % of total cross sectional area

Entrainment separator value Rd = 1.3

Vertical tube outer diameter = 100 mm

Tube thickness = 1.5 mm

Effective length of the tube = 1200 mm

Pitch of tubes (triangular) = 125 mm

Permissible stress for low carbon steel = 98 N/mm²

Modulus of elasticity for low carbon steel = 20 x 10⁴ N/mm²

Modulus of elasticity for brass = 9.6 x 10⁴ N/mm²

Bottom head cone angle = 120⁰

Number of bolts = 112

Size of bolts = M18

Factor of safety = 4

Poisson ratio = 0.33

Height of drum = 3000 mm

Design

i) Calandria (Diameter and Thickness)

ii) Vapor drum (Diameter and Thickness)

Q 6 (a) Explain process flow diagram and piping and instrumentation diagram in detail. **10**

(b) Draw symbols for **05**

i) Fixed bed reactor

ii) Pressure filter

iii) Drum drier

iv) Butterfly valve

v) Centrifugal pump

(c) Short note on Rotary Drum Filter. **05**
